

KISSsoft evaluation

File

Name : CylGearPair 6 (Helical DIN3990)  
 Description: KISSsoft example  
 Changed by: kspl on: 07.03.2016 at: 10:48:31

**CALCULATION OF A HELICAL GEAR PAIR**

Drawing or article number:

Gear 1: 0.000.0  
 Gear 2: 0.000.0

Calculation method DIN 3990:1987 Method B

----- GEAR 1 ----- GEAR 2 --

Power (kW)	[P]		12.500	
Speed (1/min)	[n]	2950.0		765.9
Torque (Nm)	[T]	40.5		155.9
Application factor	[KA]		1.25	
Required service life (h)	[H]		20000.00	
Gear driving (+) / driven (-)		+		-
Working flank gear 1: Right flank				

**1. TOOTH GEOMETRY AND MATERIAL**

(geometry calculation according to  
 DIN 3960:1987)

----- GEAR 1 ----- GEAR 2 --

Center distance (mm)	[a]		101.845	
Centre distance tolerance		ISO 286:2010 Measure js7		
Normal module (mm)	[mn]		1.5000	
Pressure angle at normal section (°)	[alfn]		20.0000	
Helix angle at reference circle (°)	[beta]		14.0000	
Number of teeth	[z]	27		104
Facewidth (mm)	[b]	21.00		20.00
Hand of gear		left		right
Accuracy grade	[Q-DIN 3961:1978]	6		6
Inner diameter (mm)	[di]	0.00		146.39
Inner diameter of gear rim (mm)	[dbi]	0.00		0.00

Material

Gear 1: 31 CrMoV9, Nitriding steel, gas-nitrided  
ISO 6336-5 Figure 13b/14b (MQ)

Gear 2: 42 CrMo 4 (3), Through hardened steel, nitrided  
ISO 6336-5 Figure 13a/14a (MQ)

		----- GEAR 1 -----	----- GEAR 2 --
Surface hardness		HV 800	HV 550
Fatigue strength. tooth root stress (N/mm <sup>2</sup> )	[sigFlim]	425.00	370.00
Fatigue strength for Hertzian pressure (N/mm <sup>2</sup> )	[sigHlim]	1250.00	1000.00
Tensile strength (N/mm <sup>2</sup> )	[Rm]	1100.00	1100.00
Yield point (N/mm <sup>2</sup> )	[Rp]	900.00	900.00
Young's modulus (N/mm <sup>2</sup> )	[E]	206000	206000
Poisson's ratio	[ny]	0.300	0.300
Roughness average value DS, flank (µm)	[RAH]	3.00	3.00
Roughness average value DS, root (µm)	[RAF]	3.00	3.00
Mean roughness height, Rz, flank (µm)	[RZH]	20.00	20.00
Mean roughness height, Rz, root (µm)	[RZF]	20.00	20.00

Gear reference profile 1 :

Reference profile	1.25 / 0.38 / 1.0 ISO 53.2:1997 Profil A	
Dedendum coefficient	[hfP*]	1.250
Root radius factor	[rhofP*]	0.380 (rhofPmax*= 0.472)
Addendum coefficient	[haP*]	1.000
Tip radius factor	[rhoaP*]	0.000
Protuberance height factor	[hprP*]	0.000
Protuberance angle	[alfprP]	0.000
Tip form height coefficient	[hFaP*]	0.000
Ramp angle	[alfKP]	0.000
		not topping

Gear reference profile 2 :

Reference profile	1.25 / 0.38 / 1.0 ISO 53.2:1997 Profil A	
Dedendum coefficient	[hfP*]	1.250
Root radius factor	[rhofP*]	0.380 (rhofPmax*= 0.472)
Addendum coefficient	[haP*]	1.000
Tip radius factor	[rhoaP*]	0.000
Protuberance height factor	[hprP*]	0.000
Protuberance angle	[alfprP]	0.000
Tip form height coefficient	[hFaP*]	0.000
Ramp angle	[alfKP]	0.000
		not topping

Summary of reference profile gears:

Dedendum reference profile	[hfP*]	1.250	1.250
Tooth root radius Refer. profile	[rofP*]	0.380	0.380
Addendum Reference profile	[haP*]	1.000	1.000
Protuberance height factor	[hprP*]	0.000	0.000
Protuberance angle (°)	[alfprP]	0.000	0.000
Tip form height coefficient	[hFaP*]	0.000	0.000
Ramp angle (°)	[alfKP]	0.000	0.000

Type of profile modification:  
for high load capacity gearboxes

Tip relief (µm)	[Ca]	5.0	5.0
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Lubrication type	oil bath lubrication		
Type of oil	Oil: Mobilgear 600 XP 150		
Lubricant base	Mineral-oil base		
Kinem. viscosity oil at 40 °C (mm <sup>2</sup> /s)	[nu40]	150.00	
Kinem. viscosity oil at 100 °C (mm <sup>2</sup> /s)	[nu100]	14.70	
FZG test A/8.3/90 (ISO 14635-1:2006)	[FZGtestA]	12	
Specific density at 15 °C (kg/dm <sup>3</sup> )	[roOil]	0.890	
Oil temperature (°C)	[TS]	70.000	

----- GEAR 1 ----- GEAR 2 --

Overall transmission ratio	[itot]	-3.852	
Gear ratio	[u]	3.852	
Transverse module (mm)	[mt]	1.546	
Pressure angle at pitch circle (°)	[alft]	20.562	
Working transverse pressure angle (°)	[alfwt]	21.425	
	[alfwt.e/i]	21.450 /	21.400
Working pressure angle at normal section (°)	[alfwn]	20.838	
Helix angle at operating pitch circle (°)	[betaw]	14.078	
Base helix angle (°)	[betab]	13.140	
Reference centre distance (mm)	[ad]	101.258	
Sum of profile shift coefficients	[Summexi]	0.3994	
Profile shift coefficient	[x]	0.3255	0.0739
Tooth thickness (Arc) (module) (module)	[sn*]	1.8077	1.6246
Tip alteration (mm)	[k*mn]	-0.012	-0.012
Reference diameter (mm)	[d]	41.740	160.776
Base diameter (mm)	[db]	39.081	150.533
Tip diameter (mm)	[da]	45.692	163.973
(mm)	[da.e/i]	45.692 /	45.682 163.973 /
163.963			
Tip diameter allowances (mm)	[Ada.e/i]	0.000 /	-0.010 0.000 /
-0.010			
Tip form diameter (mm)	[dFa]	45.692	163.973
(mm)	[dFa.e/i]	45.692 /	45.682 163.973 /

163.963				
Active tip diameter (mm)	[dNa]	45.692		163.973
Active tip diameter (mm)	[dNa.e/i]	45.692 /	45.682	163.973 /
163.963				
Operating pitch diameter (mm)	[dw]	41.982		161.708
(mm)	[dw.e/i]	41.989 /	41.975	161.736 /
161.680				
Root diameter (mm)	[df]	38.966		157.247
Generating Profile shift coefficient	[xE.e/i]	0.2852/	0.2669	0.0006/
-0.0452				
Manufactured root diameter with xE (mm)	[df.e/i]	38.845 /	38.791	157.028 /
156.890				
Theoretical tip clearance (mm)	[c]	0.375		0.375
Effective tip clearance (mm)	[c.e/i]	0.576 /	0.468	0.486 /
0.418				
Active root diameter (mm)	[dNf]	40.193		158.852
(mm)	[dNf.e/i]	40.221 /	40.171	158.888 /
158.821				
Root form diameter (mm)	[dFf]	40.081		158.171
(mm)	[dFf.e/i]	40.006 /	39.973	157.980 /
157.862				
Reserve (dNf-dFf)/2 (mm)	[cF.e/i]	0.124 /	0.082	0.513 /
0.421				
Addendum (mm)	[ha=mn*(haP*+x)]	1.976		1
.599				
(mm)	[ha.e/i]	1.976 /	1.971	1.599 /
1.594				
Dedendum (mm)	[hf=mn*(hfP*-x)]	1.387		1
.764				
(mm)	[hf.e/i]	1.447 /	1.475	1.874 /
1.943				
Roll angle at dFa (°)	[xsi_dFa.e/i]	34.709 /	34.681	24.746 /
24.736				
Roll angle to dNa (°)	[xsi_dNa.e/i]	34.709 /	34.681	24.746 /
24.736				
Roll angle to dNf (°)	[xsi_dNf.e/i]	13.943 /	13.626	19.353 /
19.272				
Roll angle at dFf (°)	[xsi_dFf.e/i]	12.542 /	12.312	18.243 /
18.095				
Tooth height (mm)	[H]	3.363		3
.363				
Virtual gear no. of teeth	[zn]	29.343		113
.025				
Normal-tooth thickness at tip circle (mm)	[san]	0.973		1
.218				
(mm)	[san.e/i]	0.931 /	0.903	1.141 /
1.086				

Normal-tooth thickness on tip form circle (mm) .218	[sFan]	0.973			1
(mm)	[sFan.e/i]	0.931 /	0.903		1.141 /
1.086					
Normal space width at root circle (mm) .077	[efn]	0.000			1
(mm)	[efn.e/i]	0.000 /	0.000		1.090 /
1.099					
Max. sliding velocity at tip (m/s) .157	[vga]	1.622			1
Specific sliding at the tip .444	[zetaa]	0.444			0
Specific sliding at the root .798	[zetaf]	-0.798			-0
Mean specific sliding	[zetam]		0.444		
Sliding factor on tip .178	[Kga]	0.250			0
Sliding factor on root .250	[Kgf]	-0.178			-0
Pitch on reference circle (mm)	[pt]		4.857		
Base pitch (mm)	[pbt]		4.547		
Transverse pitch on contact-path (mm)	[pet]		4.547		
Lead height (mm) .813	[pz]	525.932			2025
Axial pitch (mm)	[px]		19.479		
Length of path of contact (mm)	[ga, e/i]	7.142 (	7.190 /	7.072)	
Length T1-A, T2-A (mm) 32.508/	[T1A, T2A]	4.695(	4.647/	4.755)	32.508(
		32.495)			
Length T1-B (mm) 29.864/	[T1B, T2B]	7.290(	7.290/	7.280)	29.912(
		29.970)			
Length T1-C (mm) 29.497/	[T1C, T2C]	7.668(	7.658/	7.678)	29.535(
		29.573)			
Length T1-D (mm) 27.960/	[T1D, T2D]	9.242(	9.194/	9.303)	27.960(
		27.948)			
Length T1-E (mm) 25.317/	[T1E, T2E]	11.837(	11.837/	11.828)	25.365(
		25.423)			
Length T1-T2 (mm) .250)	[T1T2]		37.202 (	37.154 /	37
Diameter of single contact point B (mm) 161.950/	[d-B]	41.712(	41.712/	41.705)	161.986(
		162.028)			
Diameter of single contact point D (mm) 160.585/	[d-D]	43.232(	43.191/	43.283)	160.585(
		160.576)			
Addendum contact ratio 0.662/	[eps]	0.917(	0.919/	0.913)	0.654(
		0.643)			
Minimal length of contact line (mm)	[Lmin]		31.954		
Transverse contact ratio	[eps_a]		1.571		

Transverse contact ratio with allowances	[eps_a.e/m/i]	1.581 / 1.568 / 1.555
Overlap ratio	[eps_b]	1.027
Total contact ratio	[eps_g]	2.597
Total contact ratio with allowances	[eps_g.e/m/i]	2.608 / 2.595 / 2.582

## 2. FACTORS OF GENERAL INFLUENCE

		----- GEAR 1 -----	GEAR 2 --
Nominal circum. force at pitch circle (N)	[Ft]		1938.8
Axial force (N)	[Fa]		483.4
Radial force (N)	[Fr]		727.3
Normal force (N)	[Fnorm]		2126.4
Nominal circumferential force per mm (N/mm)	[w]		96.94
Only as information: Forces at operating pitch circle:			
Nominal circumferential force (N)	[Ftw]		1927.6
Axial force (N)	[Faw]		483.4
Radial force (N)	[Frw]		756.4
Circumferential speed reference circle (m/s)	[v]		6.45
Circumferential speed operating pitch circle (m/s)	[v(dw)]		6.48
Running-in value ( $\mu\text{m}$ )	[yp]		0.6
Running-in value ( $\mu\text{m}$ )	[yf]		0.4
Correction coefficient	[CM]		0.800
Gear body coefficient .619)	[CR, bs/b, sr/mn]		0.933 (0.500, 3
Reference profile coefficient	[CBS]		0.975
Material coefficient	[E/Est]		1.000
Singular tooth stiffness (N/mm/ $\mu\text{m}$ )	[c']		13.649
Meshing stiffness (N/mm/ $\mu\text{m}$ )	[cg]		19.491
Reduced mass (kg/mm)	[mRed]		0.00525
Resonance speed (min-1)	[nE1]		21549
Resonance ratio (-)	[N]		0.137
Subcritical range			
Running-in value ( $\mu\text{m}$ )	[ya]		0.6
Bearing distance l of pinion shaft (mm)	[l]		42.000
Distance s of pinion shaft (mm)	[s]		4.200
Outside diameter of pinion shaft (mm)	[dsh]		21.000
Load according to Figure 6.8, DIN 3990-1:1987 (0:6.8a, 1:6.8b, 2:6.8c, 3:6.8d, 4:6.8e)	[-]	4	
Coefficient K' according to Figure 6.8, DIN 3990-1:1987 Without support effect	[K']		-1.00
Tooth trace deviation (active) ( $\mu\text{m}$ )	[Fby]		4.65
from deformation of shaft ( $\mu\text{m}$ )	[fsh*B1]		0.40
(fsh ( $\mu\text{m}$ ) = 0.80, B1= 0.50, fHb5 ( $\mu\text{m}$ ) = 6.00)			

Tooth trace: width-crowned	[Cbeta = 0.5*(fma+fsh)]		
Position of Contact pattern: favorable			
from production tolerances (µm)	[fma*B2]		4.50
(B2=			
0.50)			
Tooth trace deviation, theoretical (µm)	[Fbx]		5.47
Running-in value (µm)	[yb]		0.82
Dynamic factor	[KV]		1.063
Face load factor - flank	[KHb]		1.351
- Tooth root	[KFb]		1.284
- Scuffing	[KBb]		1.351
Transverse load factor - flank	[KHα]		1.069
- Tooth root	[KFα]		1.069
- Scuffing	[KBα]		1.069
Helical load factor scuffing	[Kbg]		1.240
Number of load cycles (in mio.)	[NL]	3540.000	919.038

### 3. TOOTH ROOT STRENGTH

Calculation of Tooth form coefficients according method: B

		----- GEAR 1 -----	GEAR 2 --
Calculated with manufacturing profile shift	[xE.e]	0.29	0.00
Tooth form factor	[YF]	1.20	1.35
Stress correction factor	[YS]	2.19	2.12
Working angle (°)	[alfFen]	21.30	20.27
Bending lever arm (mm)	[hF]	1.43	1.74
Tooth thickness at root (mm)	[sFn]	3.27	3.40
Tooth root radius (mm)	[roF]	0.70	0.69
(hF* = 0.953/ 1.162 sFn* = 2.177/ 2.268 roF* = 0.466/ 0.459 dsFn = 39.393/ 157.625 alfsFn = 30.00/ 30.00)			
Contact ratio factor	[Yeps]		1.000
Helix angle factor	[Ybet]		0.883
Effective facewidth (mm)	[beff]	21.00	20.00
Nominal stress at tooth root (N/mm²)	[sigF0]	142.75	163.89
Tooth root stress (N/mm²)	[sigF]	260.31	298.87
Permissible bending stress at root of Test-gear			
Notch sensitivity factor	[YdrelT]	0.993	0.999
Surface factor	[YRrelT]	0.990	0.990

size factor (Tooth root)	[YX]	1.000	1.000
Finite life factor	[YNT]	1.000	1.000
	[YdreIT*YRreIT*YX*YNT]	0.983	0.989
Alternating bending factor (mean stress influence coefficient)	[YM]	1.000	1.000
Stress correction factor	[Yst]		2.00
Yst*sigFlim (N/mm <sup>2</sup> )	[sigFE]	850.00	740.00
Permissible tooth root stress (N/mm <sup>2</sup> )	[sigFP=sigFG/SFmin]	642.62	562.78
Limit strength tooth root (N/mm <sup>2</sup> )	[sigFG]	835.41	731.62
Required safety	[SFmin]	1.30	1.30
Safety for Tooth root stress	[SF=sigFG/sigF]	3.21	2.45
Transmittable power (kW)	[kWRating]	30.86	23.54

#### **4. SAFETY AGAINST PITTING (TOOTH FLANK)**

		----- GEAR 1 -----	GEAR 2 --
Zone factor	[ZH]		2.379
Elasticity coefficient ( $\sqrt{N/mm}$ )	[ZE]		189.812
Contact ratio factor	[Zeps]		0.798
Helix angle factor	[Zbet]		0.985
Effective facewidth (mm)	[beff]		20.00
Nominal flank pressure (N/mm <sup>2</sup> )	[sigH0]		607.16
Surface pressure at operating pitch circle (N/mm <sup>2</sup> )	[sigHw]		841.26
Single tooth contact factor	[ZB,ZD]	1.00	1.00
Flank pressure (N/mm <sup>2</sup> )	[sigHB, sigHD]	841.26	841.26
Lubrication coefficient at NL	[ZL]	0.988	0.988
Speed coefficient at NL	[ZV]	0.981	0.981
Roughness coefficient at NL	[ZR]	0.797	0.797
Work hardening factor at NL	[ZW]	1.000	1.000
Finite life factor	[ZNT]	1.000	1.000
	[ZL*ZV*ZR*ZNT]	0.772	0.772
Small no. of pittings permissible:	no		
Size factor (flank)	[ZX]	1.000	1.000
Permissible surface pressure (N/mm <sup>2</sup> )	[sigHP=sigHG/SHmin]	1016.23	812.99
Limit strength pitting (N/mm <sup>2</sup> )	[sigHG]	965.42	772.34
Required safety	[SHmin]	0.95	0.95
Safety for surface pressure at operating pitch circle	[SHw]	1.15	0.92
Safety for stress at single tooth contact	[SHBD=sigHG/sigHBD]	1.15	0.92
(Safety regarding transmittable torque)	[(SHBD)^2]	1.32	0.84
Transmittable power (kW)	[kWRating]	18.24	11.67



**4b. MICROPITTING ACCORDING TO ISO/TR 15144-1:2014**

Calculation of permissible specific film thickness

Lubricant load according to FVA Info sheet 54/7 10 (Oil: Mobilgear 600 XP 150)

Reference data FZG-C Test:

Torque (Nm)	[T1Ref]	265.1
Line load at contact point A (N/mm)	[FbbRef,A]	236.3
Oil temperature (°C)	[theOilRef]	90.0
Tooth mass temperature (°C)	[theMRef]	128.0
Contact temperature (°C)	[theBRef,A]	253.8
Lubrication gap thickness (µm)	[hRef,A]	0.043
Specific film thickness in test (µm)	[lamGFT]	0.085
Material coefficient	[WW]	1.00
Permissible specific film thickness (µm)	[lamGFP]	0.120

Interim results in accordance with ISO/TR 15144:2014

Coefficient of friction	[mym]	0.106
Lubricant factor	[XL]	1.000
Roughness factor	[XR]	1.831
Tooth mass temperature (°C)	[theM]	73.3
Tip relief factor	[XCa (A)]	1.817
Loss factor	[HV]	0.105
Equivalent Young's modulus (N/mm <sup>2</sup> )	[Er]	226374
Compressed viscosity index (m <sup>2</sup> /N)	[alf38]	0.02052
Dynamic viscosity (Ns/m <sup>2</sup> )	[etatM]	28.6
Roughness average value (µm)	[Ra]	3.0

Calculation of speeds, load distribution and flank curvature according to method B following ISO/TR 15144-1:2014

Ca taken as optimal in the calculation (0=no, 1=yes)	1	1	
Calculation at point (0:A, 1:AB, 2:B, 3:C, 4:D, 5:DE, 6:E, -1:No Point)		1	
Diameter (mm)	[dy]	40.877	162.962
Relative radius of curvature (mm)	[pred]	5.162	
Flank pressure (N/mm <sup>2</sup> )	[pH]	759.796	
Flank pressure (N/mm <sup>2</sup> )	[pdyn]	1052.759	
Minimal specific film thickness (µm)	[lamGFY]	0.067	(hY=0.200)
µm)			
Safety against micropitting	[Slam(B)]	0.557	

(For intermediate results refer to file: Micropitting\_12.tmp)

**5. STRENGTH AGAINST SCUFFING**

Calculation method according to

DIN 3990:1987

Lubrication coefficient (for lubrication type)	[XS]	1.000	
Relative structure coefficient (Scuffing)	[XWrelT]	1.500	
Thermal contact factor (N/mm/s <sup>0.5</sup> /K)	[BM]	13.780	13.780
Relevant tip relief (µm)	[Ca]	5.00	5.00
Optimal tip relief (µm)	[Ceff]	6.22	
Ca taken as optimal in the calculation (0=no, 1=yes)		1	1
Effective facewidth (mm)	[beff]	20.000	
Applicable circumferential force/facewidth (N/mm)			
	[wBt]	230.707	
Angle factor	[Xalfbet]	0.993	
(ε1:0.917, ε2:0.654)			
Flash temperature-criteria			
Tooth mass temperature (°C)	[theM-B]	83.48	
theM-B = theoil + XS*0.47*theflamax	[theflamax]	28.68	
Scuffing temperature (°C)	[theS]	522.88	
Coordinate gamma (point of highest temp.)	[Gamma]	0.277	
[Gamma.A]=-0.388 [Gamma.E]=0.544			
Highest contact temp. (°C)	[theB]	112.16	
Flash factor (°K*N <sup>-0.75</sup> *s <sup>0.5</sup> *m <sup>-0.5</sup> *mm)	[XM]	50.058	
Geometry factor	[XB]	0.128	
Load sharing factor	[XGam]	0.819	
Dynamic viscosity (mPa*s)	[etaM]	20.37 ( 83.5 °C)	
Coefficient of friction	[mym]	0.116	
Required safety	[SBmin]	2.000	
Safety factor for scuffing (flash temperature)	[SB]	10.740	
Integral temperature-criteria			
Tooth mass temperature (°C)	[theM-C]	79.83	
theM-C = theoil + XS*0.70*theflaint	[theflaint]	14.04	
Integral scuffing temperature (°C)	[theSint]	522.88	
Flash factor (°K*N <sup>-0.75</sup> *s <sup>0.5</sup> *m <sup>-0.5</sup> *mm)	[XM]	50.058	
Contact ratio factor	[Xeps]	0.244	
Dynamic viscosity (mPa*s)	[etaOil]	32.33 ( 70.0 °C)	
Mean coefficient of friction	[mym]	0.111	
Geometry factor	[XBE]	0.236	
Meshing factor	[XQ]	1.000	
Tip relief factor	[XCa]	1.068	
Integral tooth flank temperature (°C)	[theint]	100.88	
Required safety	[SSmin]	1.800	
Safety factor for scuffing (intg.-temp.)	[SSint]	5.183	
Safety referring to transferred torque	[SSL]	14.665	

## **6. MEASUREMENTS FOR TOOTH THICKNESS**

		----- GEAR 1 -----	----- GEAR 2 --
		DIN 3967 d24	DIN 3967 d25
Tooth thickness deviation			
Tooth thickness allowance (normal section) (mm)	[As.e/i]	-0.044 / -0.064	-0.080 / -0.130
Number of teeth spanned	[k]	4.000	13.000
Base tangent length (no backlash) (mm)	[Wk]	16.451	57.809
Actual base tangent length ('span') (mm)	[Wk.e/i]	16.410 / 16.391	57.734 / 57.687
(mm)	[ΔWk.e/i]	-0.041 / -0.060	-0.075 / -0.122
Diameter of contact point (mm)	[dMWk.m]	42.218	160.682
Theoretical diameter of ball/pin (mm)	[DM]	2.713	2.527
Eff. Diameter of ball/pin (mm)	[DMeff]	2.750	2.750
Theor. dim. centre to ball (mm)	[MrK]	23.341	82.625
Actual dimension centre to ball (mm)	[MrK.e/i]	23.294 / 23.273	82.522 / 82.458
Diameter of contact point (mm)	[dMMr.m]	42.672	161.251
Diametral measurement over two balls without clearance (mm)	[MdK]	46.607	165.251
Actual dimension over balls (mm)	[MdK.e/i]	46.514 / 46.471	165.045 / 164.916
Diametral measurement over rolls without clearance (mm)	[MdR]	46.681	165.251
Actual dimension over rolls (mm)	[MdR.e/i]	46.588 / 46.545	165.045 / 164.916
Chordal tooth thickness (no backlash) (mm)	[sn]	2.710	2.437
Actual chordal tooth thickness (mm)	[sn.e/i]	2.666 / 2.646	2.357 / 2.307
Reference chordal height from da.m (mm)	[ha]	2.015	1.605
Tooth thickness (Arc) (mm)	[sn]	2.712	2.437
(mm)	[sn.e/i]	2.668 / 2.648	2.357 / 2.307
Backlash free center distance (mm)	[aControl.e/i]	101.680	101.587
Backlash free center distance, allowances (mm)	[jta]	-0.165 / -0.258	
dNf.i with aControl (mm)	[dNf0.i]	39.884	158.402
Reserve (dNf0.i-dFf.e)/2 (mm)	[cF0.i]	-0.061	0.211
Tip clearance	[c0.i(aControl)]	0.227	0.177
Centre distance allowances (mm)	[Aa.e/i]	0.018 / -0.018	
Circumferential backlash from Aa (mm)	[jtw_Aa.e/i]	0.014 / -0.014	
Radial clearance (mm)	[jrw]	0.276 / 0.147	
Circumferential backlash (transverse section) (mm)	[jtw]	0.215 / 0.115	
Rotation angle when gear 1 is fixed (°)		0.1522 / 0.0814	
Normal backlash (mm)	[jnw]	0.196 / 0.105	

## **7. GEAR ACCURACY**

----- GEAR 1 ----- GEAR 2 --

According to DIN 3961:1978

Accuracy grade	[Q-DIN3961]	6	6
Profile form deviation ( $\mu\text{m}$ )	[ff]	6.00	6.00
Profile slope deviation ( $\mu\text{m}$ )	[fHa]	5.00	5.00
Total profile deviation ( $\mu\text{m}$ )	[Ff]	8.00	8.00
Helix form deviation ( $\mu\text{m}$ )	[fbf]	5.50	4.00
Helix slope deviation ( $\mu\text{m}$ )	[fHb]	9.00	8.00
Total helix deviation ( $\mu\text{m}$ )	[Fb]	10.00	9.00
Normal base pitch deviation ( $\mu\text{m}$ )	[fpe]	7.00	8.00
Single pitch deviation ( $\mu\text{m}$ )	[fp]	7.00	8.00
Adjacent pitch difference ( $\mu\text{m}$ )	[fu]	8.00	10.00
Total cumulative pitch deviation ( $\mu\text{m}$ )	[Fp]	19.00	29.00
Sector pitch deviation over z/8 pitches ( $\mu\text{m}$ )	[Fpz/8]	12.00	18.00
Runout ( $\mu\text{m}$ )	[Fr]	14.00	19.00
Tooth Thickness Variation ( $\mu\text{m}$ )	[Rs]	8.00	11.00
Single flank composite, total ( $\mu\text{m}$ )	[Fi']	22.00	30.00
Single flank composite, tooth-to-tooth ( $\mu\text{m}$ )	[fi']	10.00	11.00
Radial composite, total ( $\mu\text{m}$ )	[Fi'']	17.00	24.00
Radial composite, tooth-to-tooth ( $\mu\text{m}$ )	[fi'']	6.00	10.00

According to DIN 58405:1972 (Feinwerktechnik):

Tooth-to-tooth composite error ( $\mu\text{m}$ )	[fi''']	7.00	9.00
Composite error ( $\mu\text{m}$ )	[Fi''']	20.00	25.00
Axis alignment error ( $\mu\text{m}$ )	[fp]	17.31	17.31
Flank direction error ( $\mu\text{m}$ )	[fbeta]	5.00	5.00
Runout ( $\mu\text{m}$ )	[Trk, Fr]	21.00	28.00

Axis alignment tolerances (recommendation acc. ISO TR 10064:1992, Quality 6)

Maximum value for deviation error of axis ( $\mu\text{m}$ )	[fSigbet]	11.55 (Fb= 11.00)
Maximum value for inclination error of axes ( $\mu\text{m}$ )	[fSigdel]	23.10

## **8. ADDITIONAL DATA**

Maximal possible centre distance (eps_a=1.0)	[aMAX]	102.821	
Weight - calculated with da (kg)	[Mass]	0.270	0.671
Total weight (kg)	[Mass]	0.941	

Moment of inertia (System referenced to wheel 1):

calculation without consideration of the exact tooth shape

single gears ((da+df)/2...di) ( $\text{kg}\cdot\text{m}^2$ )	[TraeghMom]	5.145e-005	0.003132
System ((da+df)/2...di) ( $\text{kg}\cdot\text{m}^2$ )	[TraeghMom]	0.0002626	
Torsional stiffness (MNm/rad)	[cr]	0.1	2.2
Mean coeff. of friction (acc. Niemann)	[mum]	0.108	
Wear sliding coef. by Niemann	[zetw]	0.697	
Gear power loss (kW)	[PVZ]	0.141	

(Meshing efficiency (%)) [etaz] 98.869)

Indications for the manufacturing by wire cutting:

Deviation from theoretical tooth trace (μm)		[WireErr]	148.8	38.7
Permissible deviation (μm)	[Fb/2]		5.0	4.5

**9. DETERMINATION OF TOOTH FORM**

Profile and tooth trace modifications for gear 1

**Symmetric (both flanks)**

- Tip relief, linear Caa = 5.000μm LCa = 0.865\*mn dCa = 44.408mm

Profile and tooth trace modifications for gear 2

**Symmetric (both flanks)**

- Tip relief, linear Caa = 5.000μm LCa = 0.865\*mn dCa = 162.966mm

Data for the tooth form calculation :

Data not available.

**10. SERVICE LIFE, DAMAGE**

Required safety for tooth root	[SFmin]	1.30
Required safety for tooth flank	[SHmin]	0.95

Service life (calculated with required safeties):

System service life (h)	[Hatt]	35.754
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Tooth root service life (h)	[HFatt]	1e+006	1e+006
Tooth flank service life (h)	[HHatt]	1e+006	35.75

Note: The entry 1e+006 h means that the Service life > 1,000,000 h.

Damage calculated on basis of required service life

[H] ( 20000.0 h)

F1%	F2%	H1%	H2%
0.00	0.00	0.00	9999.99

Damage calculated on basis of system service life

[Hatt] ( 35.8 h)

F1%	F2%	H1%	H2%
0.00	0.00	0.00	100.00

**REMARKS:**

- Specifications with [e/i] imply: Maximum [e] and Minimal value [i] with consideration of all tolerances  
Specifications with [m] imply: Mean value within tolerance
- For the backlash tolerance, the center distance tolerances and the tooth thickness deviation are taken into account. Shown is the maximal and the minimal backlash corresponding the largest resp. the smallest allowances  
The calculation is done for the Operating pitch circle..
- Details of calculation method:
  - cg according to method B
  - KV according to method B
  - KHb, KFb according method C
  - KHa, KFa according to method B

End of Report

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lines: